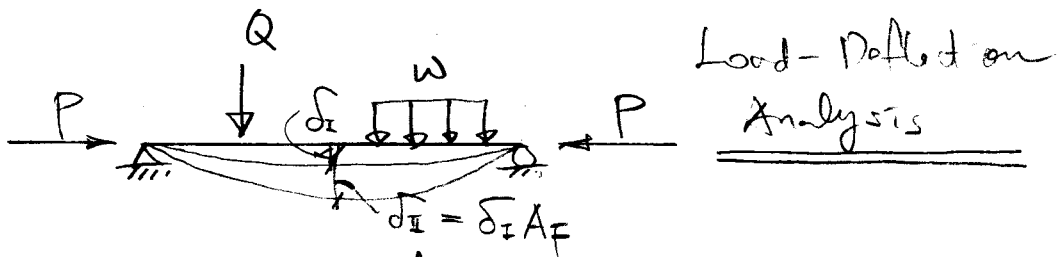
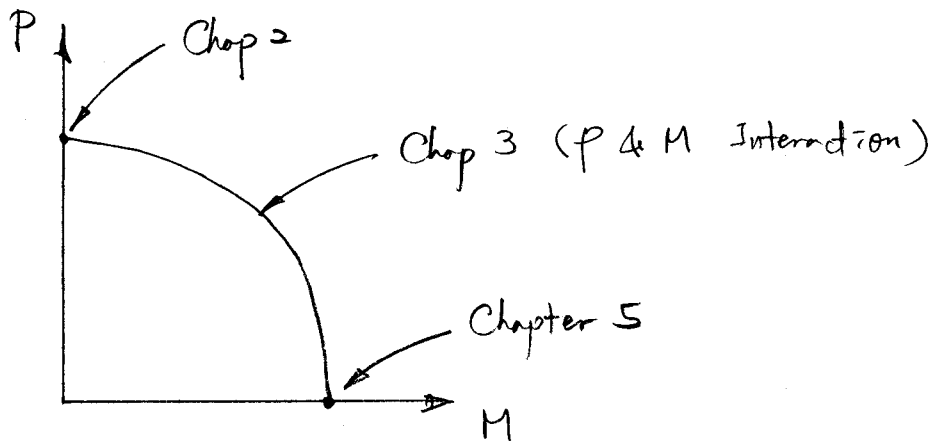


# Chapter 3: Beam-Columns

## Introduction

Beam-Column ; Compression & Bending 을 받는 부재



Load-Deflection Analysis

- 2nd-Order Deflection
- 2nd-Order Moment
- Interaction Equation

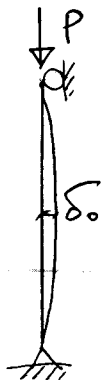
$$\sqrt{\frac{P_u}{P_n}} + \sqrt{\frac{M_u}{M_n}} = 1$$

$P_n, M_n$  ; strength

$P_u, M_u$  ; 2nd-Order Member Forces

## Amplification Factor, $A_F$

$A_F$  for Initially crooked column

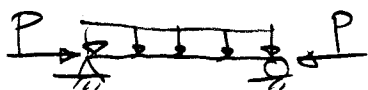


$$A_F = \frac{1}{1 - P/P_e} //$$

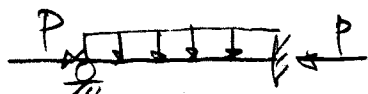
$A_F$  for beam-column

$$A_F = \frac{1 + \psi P/P_{ek}}{1 - P/P_{ek}}$$

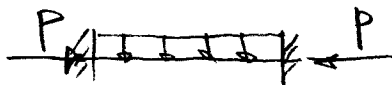
$\psi$ ; depend on boundary and loading condition



$$\psi = 0$$



$$\psi = -0.4$$



$$\psi = -0.4$$

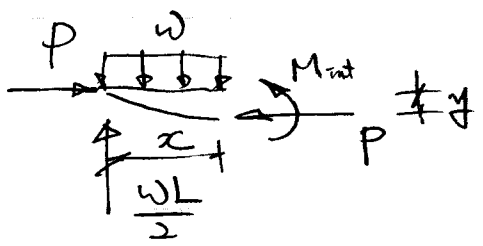
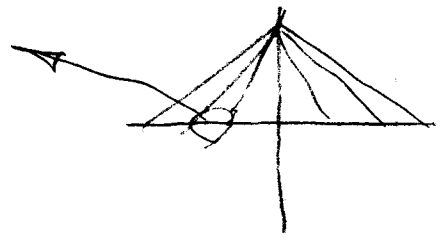
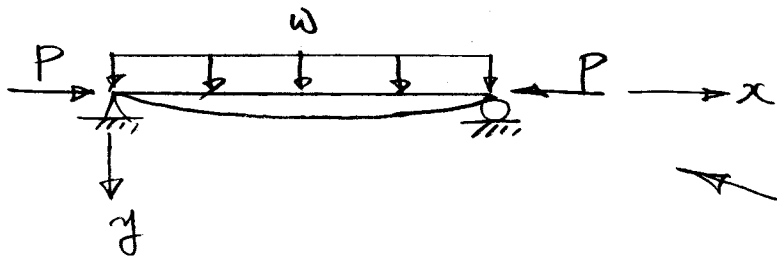


$$\psi = -0.2$$

;

;

# Simply Supported Beam-Column with Distributed Load



$$M_{int} = Py - \frac{w}{2}x^2 + \frac{wL}{2}x$$

$$M_{int} = -EIy''$$

$$EIy'' + Py = \frac{w}{2}x^2 - \frac{wL}{2}x$$

$$k^2 = \frac{P}{EI}$$

$$y'' + k^2y = \frac{w}{2EI}x^2 - \frac{wL}{2EI}x$$

$$y_h = A \sin kx + B \cos kx$$

$$y_p = C_1x^2 + C_2x + C_3$$

$$C_1 = \frac{w}{2EI k^2}, \quad C_2 = -\frac{wL}{2EI k^2}, \quad C_3 = -\frac{w}{EI k^4}$$

$$y = y_h + y_p = A \sin kx + B \cos kx + \frac{w}{2EI k^2}x^2 - \frac{wL}{2EI k^2}x - \frac{w}{EI k^4}$$

B.C

$y(0) = 0$ ,  $y'(L/2) = 0$

$$y = \frac{w}{EI k^4} \left[ \tan \frac{kL}{2} \sin kx + \cos kx - 1 \right] - \frac{w}{2EI k^2} x(L-x) //$$

$(u = \frac{kL}{2})$

$$y = \frac{wL^4}{16EI u^4} \left[ \tan u \sin \frac{2ux}{L} + \cos \frac{2ux}{L} - 1 \right] - \frac{wL^2}{8EI u^2} x(L-x)$$

$$y_{max} = \underline{\underline{y(L/2)}} = \frac{5wL^4}{384EI} \left[ \frac{12(2\sec u - u^2 - 2)}{5u^4} \right]$$

$$= y_0 \left[ \frac{12(2\sec u - u^2 - 2)}{5u^4} \right] = y_0 \frac{A_T}{}$$

(Power Series  
 $\sec u = 1 + \frac{1}{2}u^2 + \frac{5}{24}u^4 + \dots$ )

↑ Theoretical  $A_T$

$$y_{max} = y_0 [1 + 0.4067u^2 + 0.1649u^4 + \dots]$$

$(u = \frac{kL}{2} = \frac{L}{2} \sqrt{\frac{P}{EI}} = \frac{\pi}{2} \sqrt{\frac{P}{P_e}})$

$$y_{max} = y_0 \left[ 1 + 1.003 \left(\frac{P}{P_e}\right) + 1.004 \left(\frac{P}{P_e}\right)^2 + \dots \right]$$

$$\approx y_0 \left[ 1 + \left(\frac{P}{P_e}\right) + \left(\frac{P}{P_e}\right)^2 \right]$$

$$= y_0 \left[ \frac{1}{1 - P/P_e} \right] = y_0 \frac{A_T}{}$$

↑ design  $A_T$

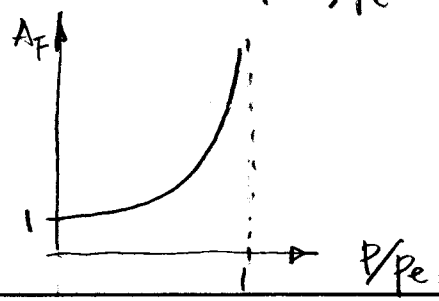


Table 3.1 (P15)

Theoretical and Design  $A_T$

**Table 3.1** Theoretical and Design Deflection Amplification Factors for a Uniformly Loaded Beam-Column

$u = \frac{kL}{2} = \frac{\pi}{2} \sqrt{\frac{P}{P_c}}$	Theoretical Eq. (3.2.30)	Design Eq. (3.2.35)
0	1.000	1.000
0.20	1.016	1.016
0.40	1.070	1.069
0.60	1.173	1.171
0.80	1.354	1.350
1.00	1.690	1.681
1.20	2.400	2.402
1.40	4.822	4.863
$\pi/2$	$\infty$	$\infty$

$$M = -EI y'' = \frac{wL^2}{4u^2} \left[ \tan u \sin \frac{2ux}{L} + \cos \frac{2ux}{L} - 1 \right]$$

$$M_{max} = M \left( \frac{L}{2} \right) = \frac{wL^2}{4u^2} [\sec u - 1]$$

$$= \frac{wL^2}{8} \left[ \frac{2(\sec u - 1)}{u^2} \right]$$

$$= M_0 \left[ \frac{2(\sec u - 1)}{u^2} \right] = M \cdot A_F$$

Theoretical  $A_F$

(Alternative

$$M_{max} = \underbrace{M_0}_{\text{primary moment}} + \underbrace{P y_{max}}_{\text{second moment}} \text{ (due to } P \text{)})$$

)

$$M_{max} = M_0 [1 + 0.4167u^2 + 0.1694u^4 + 0.06670u^6 + \dots]$$

$$= M_0 [1 + 1.028 \left(\frac{P}{P_e}\right) + 1.031 \left(\frac{P}{P_e}\right)^2 + 1.032 \left(\frac{P}{P_e}\right)^3 + \dots]$$

$$= M_0 \left\{ 1 + \left[ 1.028 \left(\frac{P}{P_e}\right) \right] \left[ 1 + 1.003 \left(\frac{P}{P_e}\right) + 1.004 \left(\frac{P}{P_e}\right)^2 + \dots \right] \right\}$$

$$\approx M_0 \left\{ 1 + \left[ 1.028 \left(\frac{P}{P_e}\right) \right] \left[ 1 + \left(\frac{P}{P_e}\right) + \left(\frac{P}{P_e}\right)^2 + \dots \right] \right\}$$

$$= M_0 \frac{1 + 0.028 P/P_e}{1 - P/P_e}$$

$$\approx M_0 \left[ \frac{1}{1 - P/P_e} \right]$$

$$= M_0 \cdot A_F \quad \text{vs.} \quad A_F = \frac{1 + \psi P/P_{e0}}{1 - P/P_{e0}} \quad (\psi = 0)$$

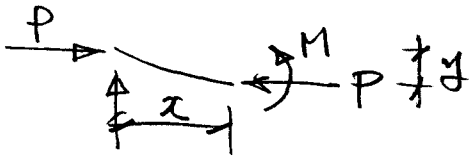
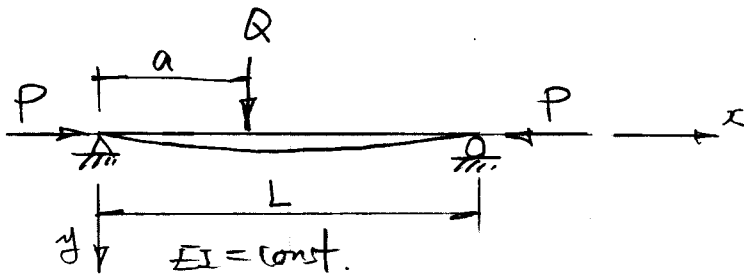
↓  
Design  $A_F$

See Table 3.2 (P157)

**Table 3.2** Theoretical and Design Moment Amplification Factors for a Uniformly Loaded Beam-Column

$u = \frac{kL}{2} = \frac{\pi}{2} \sqrt{\frac{P}{P_e}}$	Theoretical Eq. (3.2.36)	Design Eq. (3.2.41)
0	1.000	1.000
0.20	1.017	1.016
0.40	1.071	1.069
0.60	1.176	1.171
0.80	1.360	1.350
1.00	1.702	1.681
1.20	2.444	2.402
1.40	4.983	4.863
$\pi/2$	$\infty$	$\infty$

## Beam - Column with Concentrated Load



$$Q \frac{L-a}{L}$$

$$0 < x < a$$

$$a < x < L$$

$$y'' + k^2 y = \frac{-Q(L-a)}{LEI} x$$

$$y'' + k^2 y = -\frac{Qa(L-x)}{LEI}$$

$$y = A \cos kx + B \sin kx + \dots$$

$$y = C \cos kx + D \sin kx + \dots$$

$$y' =$$

$$y' =$$

B.C

$$y(0) = 0$$

$$y(L) = 0$$

C.C

$$y(a) = y(a)$$

$$y'(a) = y'(a)$$

A, B, C, D



$$y = \frac{Q}{EI k^3} \frac{\sin k(L-a)}{\sin kL} \sin kx - \frac{Q(L-a)}{LEI k^2} x \quad \text{for } 0 \leq x \leq a$$

$$y = -\frac{Q \sin ka}{EI k^3 \tan kL} \sin kx + \frac{Q \sin ka}{EI k^3} \cos kx - \frac{Qa(L-x)}{LEI k^3} \quad \text{for } a \leq x \leq L$$

$$y_{\max} \rightarrow a = \frac{L}{2} \quad \text{at } x = \frac{L}{2}$$

$$y_{\max} = \frac{QL^3}{48EI} \left[ \frac{3(\tan u - u)}{u^3} \right] = y_0 \left[ \frac{3(\tan u - u)}{u^3} \right]$$

$$\left( \tan u = u + \frac{1}{3}u^3 + \frac{2}{15}u^5 + \dots \right)$$

$$y_{\max} \approx y_0 \left[ \frac{1}{1 - P/P_e} \right] \approx y_0 \boxed{A_F} \quad \text{See Table 3.3}$$

$$M_{\max} = -EI y'' = \frac{QL}{4} \left[ \frac{\tan u}{u} \right] = M_0 \left[ \frac{\tan u}{u} \right]$$

$$\approx M_0 \left[ \frac{1 - 0.10(P/P_e)}{1 - P/P_e} \right] = M_0 \frac{1 - 0.2(P/P_e)}{1 - P/P_e} = M_0 \boxed{A_F}$$

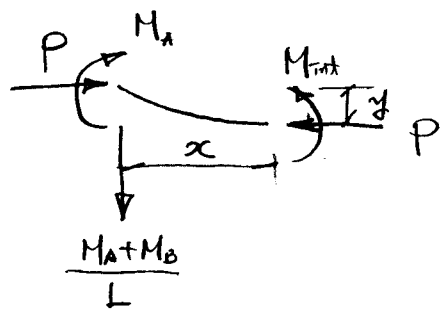
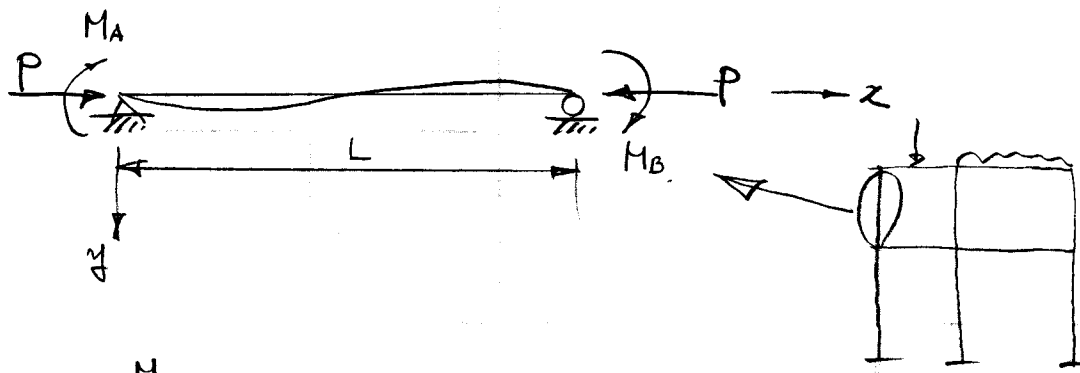
$$(\phi = -0.2)$$

See Table 3.4 (P161)

**Table 3.4** Theoretical and Design Moment Amplification Factors for a Beam-Column with a Concentrated Lateral Load at Midspan

$u = \frac{kL}{2} = \frac{\pi}{2} \sqrt{\frac{P}{P_c}}$	Theoretical Eq. (3.3.17)	Design Eq. (3.3.20)
0	1.000	1.000
0.20	1.014	1.013
0.40	1.057	1.055
0.60	1.140	1.137
0.80	1.287	1.280
1.00	1.557	1.545
1.20	2.143	2.122
1.40	4.141	4.090
$\pi/2$	$\infty$	$\infty$

Beam-Column with End Moments



$$M_{ext} = M_x + Py - \frac{M_A + M_B}{L} x$$

$$M_{int} = -EIy''$$

$$EIy'' + Py = \frac{M_A + M_B}{L} x - M_A$$

$$y'' + k^2 y = \frac{M_A + M_B}{LEI} x - \frac{M_A}{EI}$$

$$y = A \sin kx + B \cos kx + \frac{M_A + M_B}{LEIk^2} x - \frac{M_A}{EI k^2}$$

B.C

$$y(0) = 0, \quad y(L) = 0 \quad \Rightarrow \quad A, B$$

$$y = - \frac{(M_A \cos kL + M_B)}{EI k^2 \sin kL} \sin kx + \frac{M_A}{EI k^2} \cos kx + \frac{M_A + M_B}{LEIk^2} x - \frac{M_A}{EI k^2}$$

For the location of maximum moment

$$\Rightarrow \underline{\underline{EIy'' = 0}}$$

$$\tan \bar{x} = \frac{M_A \cos \bar{x}L + M_B}{-M_A \sin \bar{x}L}$$

$$\bar{x} = \square + n\pi, \quad n=0,1,\dots$$

$$\sin \bar{x} = \frac{M_A \cos \bar{x}L + M_B}{\sqrt{M_A^2 + 2M_A M_B \cos \bar{x}L + M_B^2}}$$

$$\cos \bar{x} = \frac{-M_A \sin \bar{x}L}{\sqrt{M_A^2 + 2M_A M_B \cos \bar{x}L + M_B^2}}$$

$$M_{max} = -EI y''(\bar{x})$$

$$= \frac{-(M_A \cos \bar{x}L + M_B)^2}{\sin \bar{x}L \sqrt{M_A^2 + 2M_A M_B \cos \bar{x}L + M_B^2}} - \frac{M_A^2 \sin \bar{x}L}{\sqrt{M_A^2 + 2M_A M_B \cos \bar{x}L + M_B^2}}$$

$$= - \frac{\sqrt{M_A^2 + 2M_A M_B \cos \bar{x}L + M_B^2}}{\sin \bar{x}L}$$

(Assume  $M_B > M_A$ )

$$M_{max} = -M_B \left[ \frac{(M_B/M_A)^2 + 2(M_B/M_A) \cos \bar{x}L + 1}{\sin^2 \bar{x}L} \right]$$

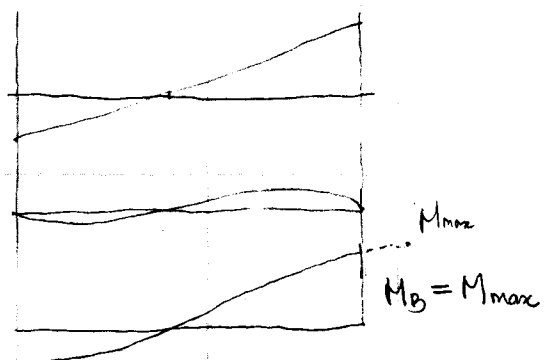
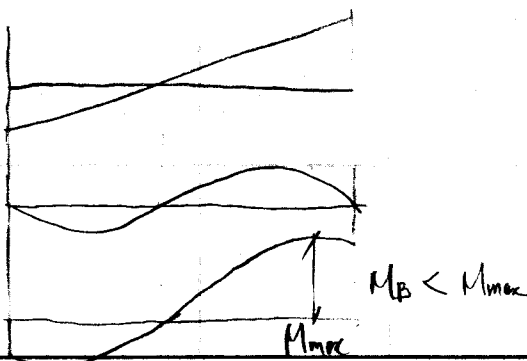
$$= -M_B A_F$$

↑  
Top side in tension

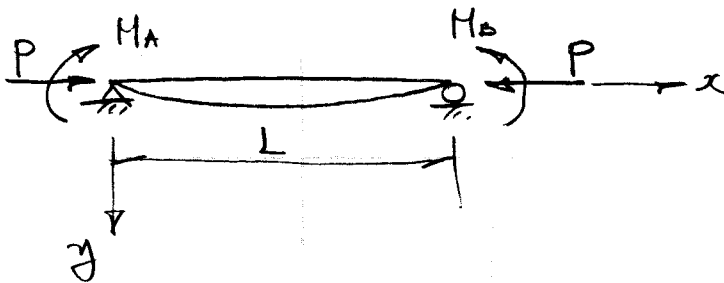
Location

$$0 < \bar{x} < L$$

$$\bar{x} < 0, \bar{x} > L$$



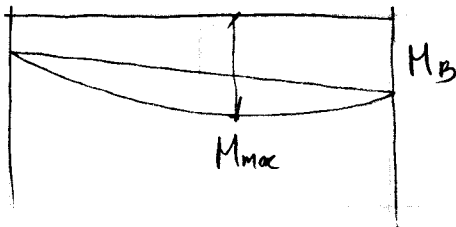
For single curvature



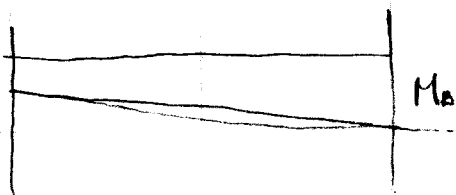
$$M_B \Rightarrow -M_B$$

$$M_{max} = M_B \left[ \frac{(M_A/M_B)^2 - 2(M_A/M_B) \cos kL + 1}{\sin^2 kL} \right]$$

$$\tan k\bar{x} = \frac{M_A \cos kL - M_B}{-M_A \sin kL}$$



$$M_{max} > M_B ; 0 < \bar{x} < L$$



$$M_B = M_{max} ; \bar{x} < 0, \bar{x} > L$$

Combined Equation

$$M_{max} = |M_B| \left[ \frac{(M_A/M_B)^2 + 2(M_A/M_B) \cos kL + 1}{\sin^2 kL} \right]$$

$M_A/M_B$  : positive in double curvature bending

$M_A/M_B$  : negative in single " "

# Concept of Equivalent Moment

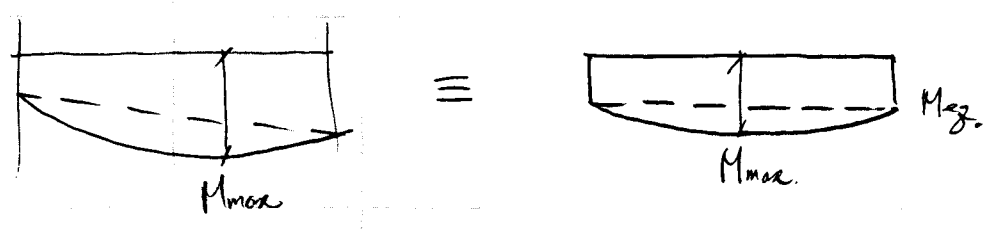
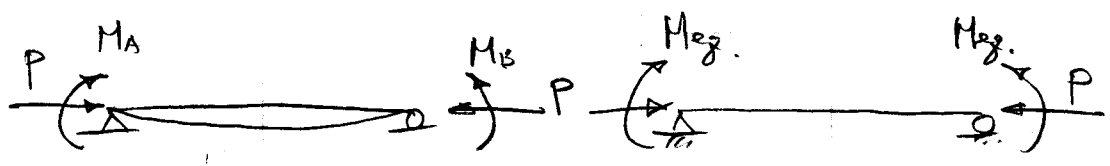
$M_{max}$  을 구하기 위해서

→  $M_{max}$  의 발생 위치 확인 필요.

$0 < \bar{x} < L$  →  $M_{max}$  from equation.

$\bar{x} < 0, \bar{x} > L$  →  $M_{max} = M_B$

이러한 계산을 생략하기 위하여  $M_{eq}$  개념 도입



$$|M_B| \left[ \frac{(M_A/M_B)^2 + 2(M_A/M_B) \cos kL + 1}{\sin^2 kL} \right] = M_{eq} \left[ \frac{2(1 - \cos kL)}{\sin^2 kL} \right]$$

$$M_{eq} = \left[ \frac{(M_A/M_B)^2 + 2(M_A/M_B) \cos kL + 1}{2(1 - \cos kL)} \right] |M_B|$$

$$= C_m |M_B|$$

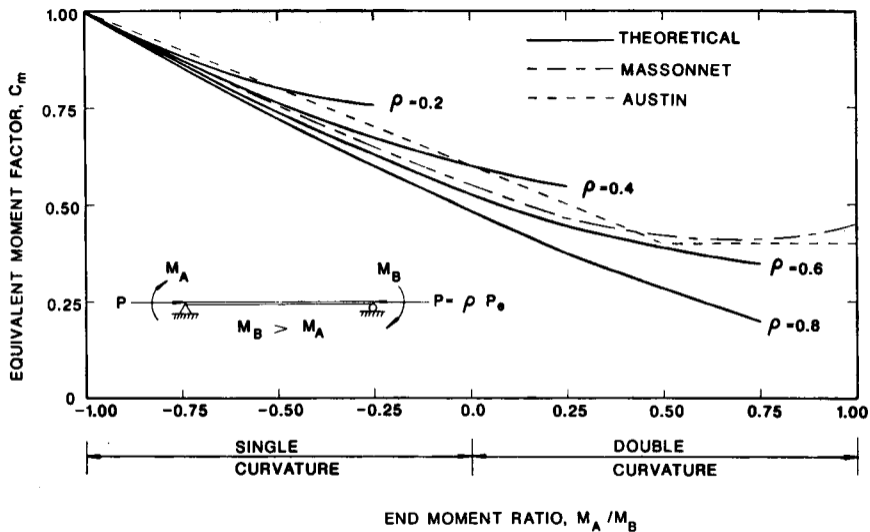
$C_m$ : equivalent moment factor,  $f(M_A/M_B, P)$

Neglect P effect on  $C_m$

$$C_m = \sqrt{0.3(M_A/M_B)^2 - 0.4(M_A/M_B) + 0.3} \quad ; \text{Massonnet}$$

$$C_m = 0.6 - 0.4(M_A/M_B) \geq 0.4 \quad ; \text{Austin}$$

See Fig. 3.11 (p169)



**FIGURE 3.11** Comparison of various expressions for  $C_m$

Maximum Moment

$$M_{max} = M_{ej} \cdot \sqrt{\frac{2(1 - \cos kL)}{\sin^2 kL}}$$

$$\left( \begin{array}{l} 1 - \cos kL = 2 \sin^2 \frac{kL}{2} \\ \sin^2 kL = 4 \sin^2 \frac{kL}{2} \cos^2 \frac{kL}{2} \end{array} \right)$$

$$M_{max} = M_{ej} \cdot \sec \frac{kL}{2} \approx M_{ej} \left( \frac{1}{1 - \frac{P}{P_e}} \right)$$

$$= \frac{C_m}{1 - \frac{P}{P_e}} \cdot M_B$$

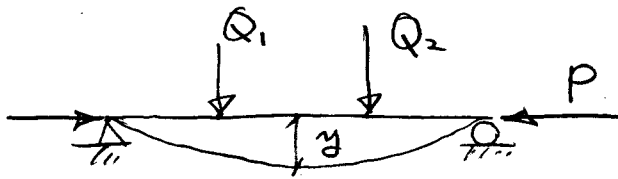
$$= A_F \cdot M_B$$

$$A_F \geq 1.0 \quad M_{max} = A_F \cdot M_B$$

$$A_F < 1.0 \quad M_{max} = M_B$$

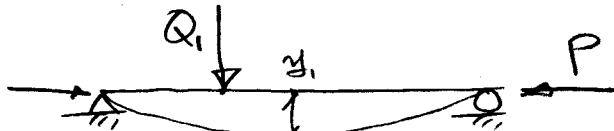


# Superposition of Solution



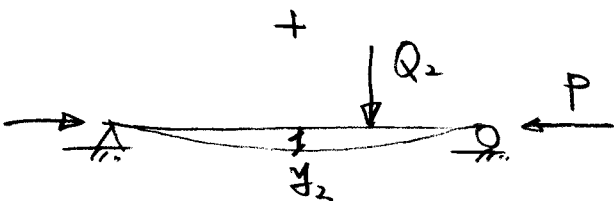
If  $P$  is constant

$$y = y_1 + y_2$$



$P = \text{constant}$

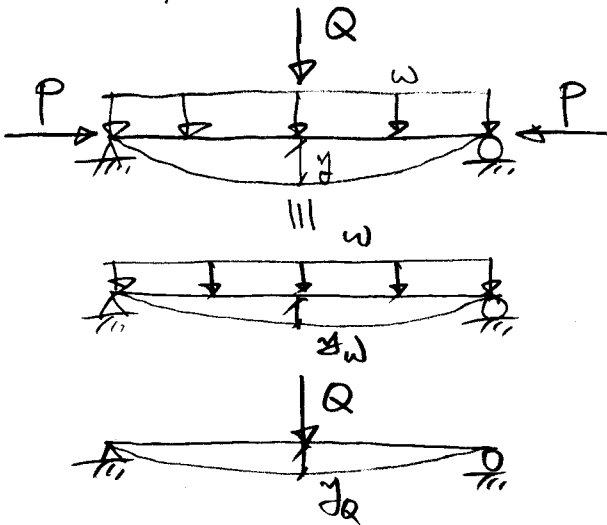
→  $Q-y$  : linear



$Q = \text{constant}$

→  $P-y$  : nonlinear

## Example

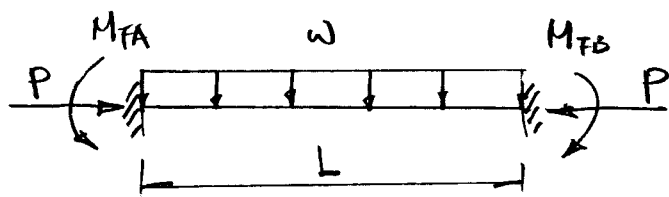


$$y = y_w + y_Q$$

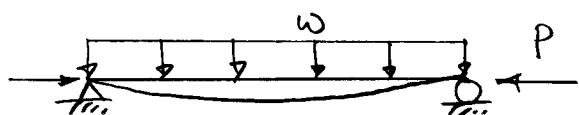
$$M_{max} = \frac{wL^2}{8} \left[ \frac{2(\sec u - 1)}{u^2} \right] + \frac{QL}{4} \left[ \frac{\tan u}{u} \right]$$

## Application of Superposition

Determine Fixed End Moment

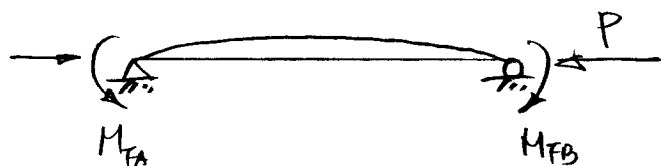


|||



$$y'_w(0) = \frac{wL^3}{24EI} \frac{3(\tan u - u)}{u^3}$$

+

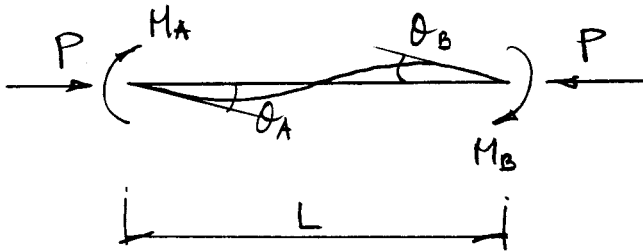


$$y'_m(0) = \frac{M_{FA} L}{2EI} \frac{\tan u}{u}$$

$$y'_w(0) = y'_m(0)$$

$$M_{FA} = \frac{wL^2}{12} \left[ \frac{3(\tan u - u)}{u^2 \tan u} \right]$$

## Slope - Deflection Eq.



For  $P=0$

$$M_A = \frac{EI}{L} (4\theta_A + 2\theta_B)$$

$$M_B = \frac{EI}{L} (2\theta_A + 4\theta_B)$$

For  $P \neq 0$  ; slope - deflection eq.

$$y = - \frac{(M_A \cos kL + M_B)}{EI k^2 \sin kL} \sin kx + \frac{M_A}{EI k^2} \cos kx + \frac{M_A + M_B}{LEI k^2} x - \frac{M_A}{EI k^2}$$

$$= - \frac{1}{EI k^2} \left[ \frac{\cos kL}{\sin kL} \sin kx - \cos kx - \frac{x}{L} + 1 \right] M_A$$

$$- \frac{1}{EI k^2} \left[ \frac{1}{\sin kL} \sin kx - \frac{x}{L} \right] M_B$$

$$y' = - \frac{1}{EI k} \left[ \frac{\cos kL}{\sin kL} \cos kx + \sin kx - \frac{1}{kL} \right] M_A$$

$$- \frac{1}{EI k} \left[ \frac{\cos kx}{\sin kL} - \frac{1}{kL} \right] M_B$$

$$\theta_A = y'(0) = \frac{L}{EI} \left[ \frac{\sin kL - kL \cos kL}{(kL)^2 \sin kL} \right] M_A + \frac{L}{EI} \left[ \frac{\sin kL - kL}{(kL)^2 \sin kL} \right] M_B$$

$$\theta_B = y'(L) = \frac{L}{EI} \left[ \frac{\sin kL - kL}{(kL)^2 \sin kL} \right] M_A + \frac{L}{EI} \left[ \frac{\sin kL - kL \cos kL}{(kL)^2 \sin kL} \right] M_B$$

$$\begin{Bmatrix} \theta_A \\ \theta_B \end{Bmatrix} = \begin{bmatrix} f_{11} & f_{12} \\ f_{21} & f_{22} \end{bmatrix} \begin{Bmatrix} M_A \\ M_B \end{Bmatrix}$$

$$f_{11} = f_{22} = \frac{L}{EI} \left[ \frac{\sin kL - kL \cos kL}{(kL)^2 \sin kL} \right]$$

$$f_{12} = f_{21} = \frac{L}{EI} \left[ \frac{\sin kL - kL}{(kL)^2 \sin kL} \right]$$

$$\begin{Bmatrix} M_A \\ M_B \end{Bmatrix} = \begin{bmatrix} f_{11} & f_{12} \\ f_{21} & f_{22} \end{bmatrix}^{-1} \begin{Bmatrix} \theta_A \\ \theta_B \end{Bmatrix}$$

$$= \frac{EI}{L} \begin{bmatrix} S_{11} & S_{12} \\ S_{21} & S_{22} \end{bmatrix} \begin{Bmatrix} \theta_A \\ \theta_B \end{Bmatrix}$$

$S_{11}, S_{12}, S_{21}, S_{22}$  : Stability function

$$S_{11} = S_{22} = \frac{kL \sin kL - (kL)^2 \cos kL}{2 - 2 \cos kL - kL \sin kL}$$

$$S_{12} = S_{21} = \frac{(kL)^2 - kL \sin kL}{2 - 2 \cos kL - kL \sin kL}$$

Table 3.7 (P185)

$kL = 0$	$P/P_e = 0$	$S_{11} = 4.0$	$S_{12} = 2$	
$kL = 3.15$	$P/P_e = 1$	$S_{11} = 2.96$	$S_{12} = 2.97$	$K = 1.0$

Fig 3.10 (P188)

**Table 3.7** Stability Functions ( $kL = \pi\sqrt{P/P_c}$ )

$kL$	$P/P_c$	compression		tension	
		$s_{ii}$	$s_{ij}$	$s_{ii}$	$s_{ij}$
0.	0.	4. 0000	2. 0000	4. 0000	2. 0000
0. 0500	0. 0003	3. 9997	2. 0001	4. 0003	1. 9999
0. 1000	0. 0010	3. 9987	2. 0003	4. 0013	1. 9997
0. 1500	0. 0023	3. 9970	2. 0008	4. 0030	1. 9993
0. 2000	0. 0041	3. 9947	2. 0013	4. 0053	1. 9987
0. 2500	0. 0063	3. 9917	2. 0021	4. 0083	1. 9979
0. 3000	0. 0091	3. 9876	2. 0028	4. 0120	1. 9970
0. 3500	0. 0124	3. 9833	2. 0039	4. 0157	1. 9956
0. 4000	0. 0162	3. 9786	2. 0054	4. 0211	1. 9946
0. 4500	0. 0205	3. 9729	2. 0068	4. 0268	1. 9932
0. 5000	0. 0253	3. 9665	2. 0084	4. 0332	1. 9917
0. 5500	0. 0306	3. 9595	2. 0102	4. 0401	1. 9900
0. 6000	0. 0365	3. 9517	2. 0121	4. 0477	1. 9881
0. 6500	0. 0428	3. 9433	2. 0143	4. 0560	1. 9861
0. 7000	0. 0496	3. 9342	2. 0166	4. 0649	1. 9839
0. 7500	0. 0570	3. 9244	2. 0191	4. 0744	1. 9816
0. 8000	0. 0648	3. 9139	2. 0218	4. 0846	1. 9791
0. 8500	0. 0732	3. 9027	2. 0246	4. 0954	1. 9764
0. 9000	0. 0821	3. 8908	2. 0277	4. 1069	1. 9737
0. 9500	0. 0914	3. 8782	2. 0309	4. 1189	1. 9707
1. 0000	0. 1013	3. 8649	2. 0344	4. 1316	1. 9677
1. 0500	0. 1117	3. 8508	2. 0380	4. 1449	1. 9645
1. 1000	0. 1226	3. 8360	2. 0419	4. 1588	1. 9611
1. 1500	0. 1340	3. 8205	2. 0460	4. 1734	1. 9577
1. 2000	0. 1459	3. 8043	2. 0502	4. 1885	1. 9541
1. 2500	0. 1583	3. 7873	2. 0547	4. 2042	1. 9503
1. 3000	0. 1712	3. 7695	2. 0594	4. 2205	1. 9465
1. 3500	0. 1847	3. 7510	2. 0644	4. 2374	1. 9425
1. 4000	0. 1986	3. 7317	2. 0695	4. 2549	1. 9384
1. 4500	0. 2130	3. 7116	2. 0749	4. 2729	1. 9342
1. 5000	0. 2280	3. 6907	2. 0806	4. 2916	1. 9299
1. 5500	0. 2434	3. 6690	2. 0865	4. 3107	1. 9255
1. 6000	0. 2594	3. 6466	2. 0926	4. 3305	1. 9210
1. 6500	0. 2758	3. 6233	2. 0990	4. 3508	1. 9163
1. 7000	0. 2928	3. 5991	2. 1057	4. 3716	1. 9116
1. 7500	0. 3103	3. 5741	2. 1127	4. 3929	1. 9068
1. 8000	0. 3283	3. 5483	2. 1199	4. 4148	1. 9019
1. 8500	0. 3468	3. 5216	2. 1275	4. 4373	1. 8969
1. 9000	0. 3658	3. 4940	2. 1353	4. 4602	1. 8919
1. 9500	0. 3853	3. 4655	2. 1434	4. 4836	1. 8867
2. 0000	0. 4053	3. 4361	2. 1519	4. 5076	1. 8815
2. 0500	0. 4258	3. 4058	2. 1607	4. 5320	1. 8762
2. 1000	0. 4468	3. 3745	2. 1699	4. 5569	1. 8708
2. 1500	0. 4684	3. 3422	2. 1794	4. 5823	1. 8654
2. 2000	0. 4904	3. 3090	2. 1893	4. 6082	1. 8599
2. 2500	0. 5129	3. 2748	2. 1996	4. 6345	1. 8544
2. 3000	0. 5360	3. 2395	2. 2102	4. 6613	1. 8488

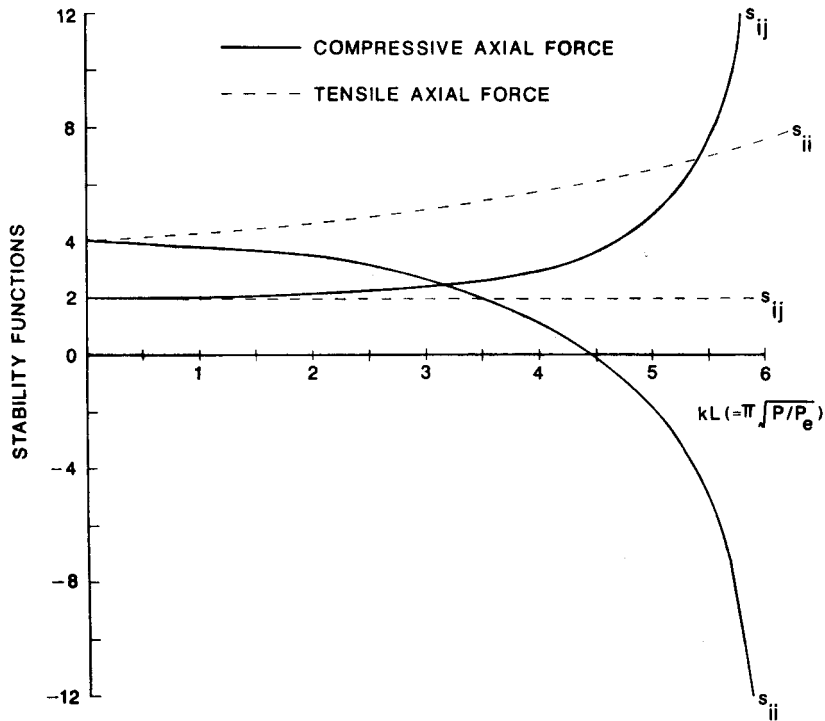
(continued)

**Table 3.7** Stability Functions ( $kL = \pi\sqrt{P/P_c}$ ) (continued)

$kL$	$P/P_c$	compression		tension	
		$s_{ii}$	$s_{ij}$	$s_{ii}$	$s_{ij}$
2. 3500	0. 5595	3. 2032	2. 2213	4. 6886	1. 8431
2. 4000	0. 5836	3. 1659	2. 2328	4. 7163	1. 8374
2. 4500	0. 6082	3. 1274	2. 2447	4. 7444	1. 8317
2. 5000	0. 6333	3. 0878	2. 2572	4. 7730	1. 8259
2. 5500	0. 6588	3. 0471	2. 2701	4. 8020	1. 8201
2. 6000	0. 6849	3. 0052	2. 2834	4. 8314	1. 8142
2. 6500	0. 7115	2. 9622	2. 2974	4. 8612	1. 8083
2. 7000	0. 7386	2. 9179	2. 3118	4. 8915	1. 8024
2. 7500	0. 7662	2. 8723	2. 3268	4. 9221	1. 7965
2. 8000	0. 7944	2. 8254	2. 3425	4. 9531	1. 7905
2. 8500	0. 8230	2. 7772	2. 3587	4. 9845	1. 7845
2. 9000	0. 8521	2. 7276	2. 3756	5. 0162	1. 7785
2. 9500	0. 8817	2. 6766	2. 3932	5. 0484	1. 7725
3. 0000	0. 9119	2. 6242	2. 4115	5. 0809	1. 7665
3. 0500	0. 9425	2. 5703	2. 4305	5. 1137	1. 7605
3. 1000	0. 9737	2. 5148	2. 4503	5. 1469	1. 7544
3. 1500	1. 0054	2. 4577	2. 4709	5. 1805	1. 7484
3. 2000	1. 0375	2. 3990	2. 4924	5. 2143	1. 7424
3. 2500	1. 0702	2. 3385	2. 5148	5. 2485	1. 7363
3. 3000	1. 1034	2. 2763	2. 5382	5. 2831	1. 7303
3. 3500	1. 1371	2. 2122	2. 5626	5. 3179	1. 7243
3. 4000	1. 1713	2. 1463	2. 5880	5. 3530	1. 7183
3. 4500	1. 2060	2. 0783	2. 6146	5. 3885	1. 7123
3. 5000	1. 2412	2. 0083	2. 6424	5. 4242	1. 7063
3. 5500	1. 2769	1. 9362	2. 6714	5. 4603	1. 7003
3. 6000	1. 3131	1. 8618	2. 7017	5. 4966	1. 6944
3. 6500	1. 3498	1. 7851	2. 7335	5. 5332	1. 6884
3. 7000	1. 3871	1. 7060	2. 7668	5. 5701	1. 6825
3. 7500	1. 4248	1. 6243	2. 8016	5. 6073	1. 6766
3. 8000	1. 4631	1. 5400	2. 8382	5. 6447	1. 6708
3. 8500	1. 5018	1. 4528	2. 8765	5. 6823	1. 6649
3. 9000	1. 5411	1. 3627	2. 9168	5. 7203	1. 6591
3. 9500	1. 5809	1. 2696	2. 9592	5. 7584	1. 6533
4. 0000	1. 6211	1. 1731	3. 0037	5. 7968	1. 6476
4. 0500	1. 6619	1. 0733	3. 0507	5. 8355	1. 6419
4. 1000	1. 7032	0. 9698	3. 1001	5. 8744	1. 6362
4. 1500	1. 7450	0. 8624	3. 1523	5. 9135	1. 6305
4. 2000	1. 7873	0. 7510	3. 2074	5. 9528	1. 6249
4. 2500	1. 8301	0. 6353	3. 2656	5. 9923	1. 6193
4. 3000	1. 8734	0. 5149	3. 3273	6. 0321	1. 6138
4. 3500	1. 9172	0. 3897	3. 3926	6. 0720	1. 6083
4. 4000	1. 9616	0. 2592	3. 4619	6. 1122	1. 6028
4. 4500	2. 0064	0. 1231	3. 5356	6. 1526	1. 5974
4. 5000	2. 0518	-0. 0191	3. 6140	6. 1931	1. 5920
4. 5500	2. 0976	-0. 1678	3. 6975	6. 2339	1. 5867
4. 6000	2. 1440	-0. 3234	3. 7866	6. 2748	1. 5814
4. 6500	2. 1908	-0. 4867	3. 8819	6. 3159	1. 5761

Table 3.7 (continued)

<i>kL</i>	<i>P/P<sub>c</sub></i>	compression		tension	
		<i>s<sub>ii</sub></i>	<i>s<sub>ij</sub></i>	<i>s<sub>ii</sub></i>	<i>s<sub>ij</sub></i>
4. 7000	2. 2382	-0. 6582	3. 9839	6. 3572	1. 5709
4. 7500	2. 2861	-0. 8387	4. 0934	6. 3987	1. 5658
4. 8000	2. 3344	-1. 0289	4. 2112	6. 4403	1. 5606
4. 8500	2. 3833	-1. 2299	4. 3381	6. 4821	1. 5556
4. 9000	2. 4327	-1. 4427	4. 4751	6. 5241	1. 5505
4. 9500	2. 4826	-1. 6685	4. 6235	6. 5662	1. 5456
5. 0000	2. 5330	-1. 9087	4. 7845	6. 6085	1. 5406
5. 0500	2. 5839	-2. 1651	4. 9599	6. 6509	1. 5357
5. 1000	2. 6354	-2. 4394	5. 1514	6. 6934	1. 5309
5. 1500	2. 6873	-2. 7341	5. 3613	6. 7362	1. 5261
5. 2000	2. 7397	-3. 0516	5. 5921	6. 7790	1. 5213
5. 2500	2. 7927	-3. 3953	5. 8470	6. 8220	1. 5166
5. 3000	2. 8461	-3. 7689	6. 1297	6. 8652	1. 5120
5. 3500	2. 9001	-4. 1770	6. 4447	6. 9084	1. 5074
5. 4000	2. 9545	-4. 6254	6. 7977	6. 9518	1. 5028
5. 4500	3. 0095	-5. 1210	7. 1957	6. 9953	1. 4983
5. 5000	3. 0650	-5. 6727	7. 6472	7. 0390	1. 4938
5. 5500	3. 1209	-6. 2916	8. 1635	7. 0827	1. 4894
5. 6000	3. 1774	-6. 9923	8. 7589	7. 1266	1. 4851
5. 6500	3. 2344	-7. 7937	9. 4524	7. 1706	1. 4807
5. 7000	3. 2919	-8. 7215	10. 2693	7. 2147	1. 4765
5. 7500	3. 3499	-9. 8106	11. 2447	7. 2590	1. 4722
5. 8000	3. 4084	-11. 1107	12. 4279	7. 3033	1. 4680
5. 8500	3. 4675	-12. 6943	13. 8915	7. 3477	1. 4639
5. 9000	3. 5270	-14. 6717	15. 7455	7. 3922	1. 4598
5. 9500	3. 5870	-17. 2192	18. 1662	7. 4369	1. 4558
6. 0000	3. 6476	-20. 6379	21. 4544	7. 4816	1. 4518
6. 0500	3. 7086	-25. 4868	26. 1690	7. 5264	1. 4478
6. 1000	3. 7702	-32. 9355	33. 4794	7. 5714	1. 4439
6. 1500	3. 8322	-45. 9092	46. 3106	7. 6164	1. 4401
6. 2000	3. 8948	-74. 3671	74. 6217	7. 6615	1. 4363
6. 2500	3. 9579	-188. 3001	188. 4032	7. 7067	1. 4325

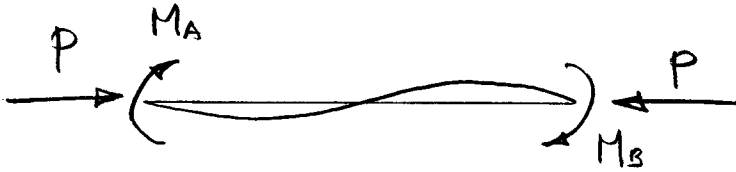


**FIGURE 3.18** Plot of stability functions



## Stability Function vs. FEM

### Stability Function



One element can predict second-order (geometrical nonlinear) effect.

PS effect 를 고려한  $M_A, M_B$  계산

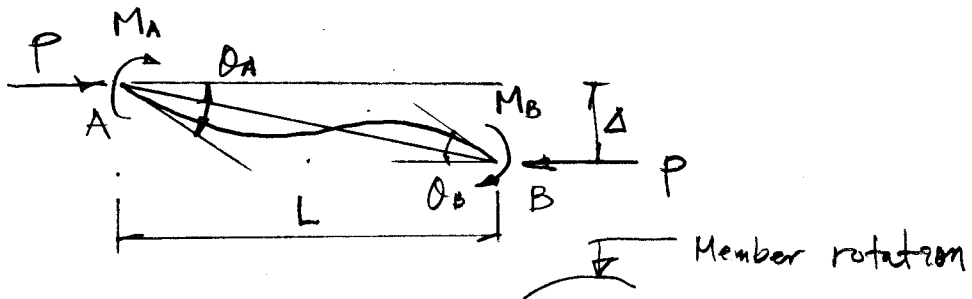
### FEM



A lot of elements are required to predict PS effect.

## Modified Slope-Deflection Equation.

1) Member with Sway



$$M_A = \frac{EI}{L} \left[ S_{ii} \left( \theta_A - \frac{\Delta}{L} \right) + S_{ij} \left( \theta_B - \frac{\Delta}{L} \right) \right]$$

$$M_B = \frac{EI}{L} \left[ S_{ji} \left( \theta_A - \frac{\Delta}{L} \right) + S_{jj} \left( \theta_B - \frac{\Delta}{L} \right) \right]$$

2) Member with a Hinge at One End (B end)

$$M_B = \frac{EI}{L} (S_{ij} \theta_A + S_{ii} \theta_B) = 0$$

$$\theta_B = - \frac{S_{ij}}{S_{ii}} \theta_A$$

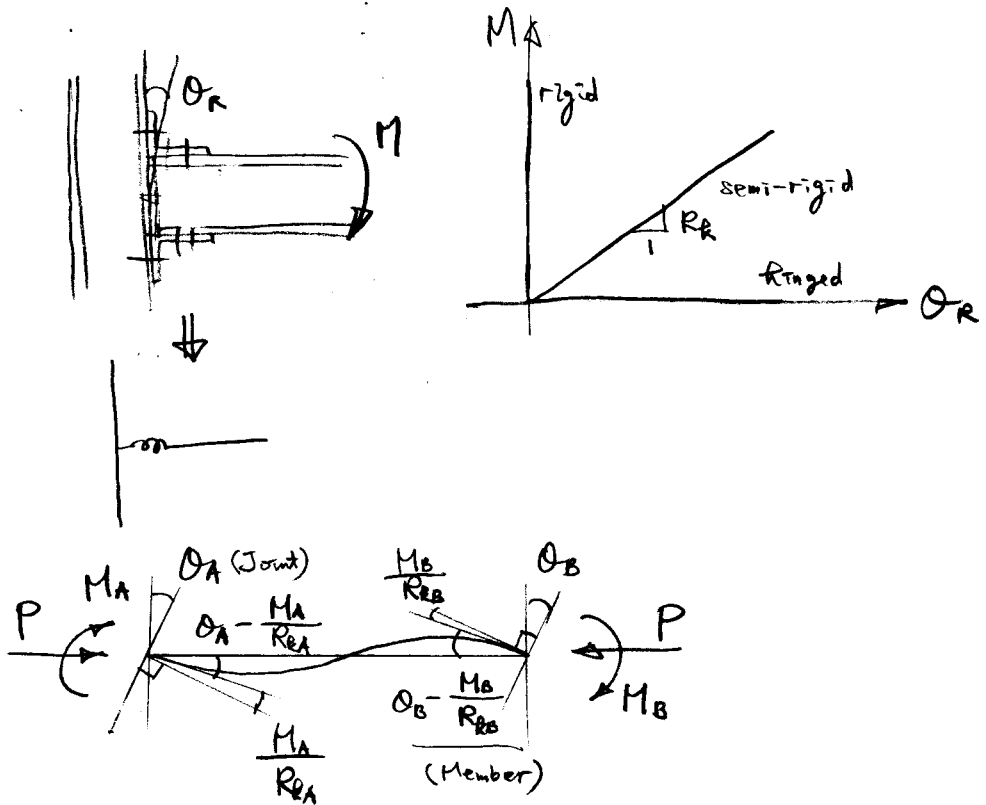
$$M_A = \frac{EI}{L} \left( S_{ii} - \frac{S_{ij}^2}{S_{ii}} \right) \theta_A$$

3) Member with Transverse Loading

$$M_A = \frac{EI}{L} (S_{ii} \theta_A + S_{ij} \theta_B) + M_{FA}$$

$$M_B = \frac{EI}{L} (S_{ji} \theta_A + S_{jj} \theta_B) + M_{FB}$$

### 4) Member with Elastically Restrained Ends



$$M_A = \frac{EI}{L} \left[ S_{ii} \left( \theta_A - \frac{M_A}{R_{EA}} \right) + S_{ij} \left( \theta_B - \frac{M_B}{R_{EB}} \right) \right]$$

$$M_B = \frac{EI}{L} \left[ S_{ij} \left( \theta_A - \frac{M_A}{R_{EA}} \right) + S_{jj} \left( \theta_B - \frac{M_B}{R_{EB}} \right) \right]$$

$$M_A = \frac{EI}{LR^*} \left[ \left( S_{ii} + \frac{EIS_{ii}^2}{LR_{EB}} - \frac{EIS_{ij}^2}{LR_{EB}} \right) \theta_A + S_{ij} \theta_B \right]$$

$$M_B = \frac{EI}{LR^*} \left[ S_{ij} \theta_A + \left( S_{jj} + \frac{EIS_{jj}^2}{LR_{EA}} - \frac{EIS_{ij}^2}{LR_{EA}} \right) \theta_B \right]$$

where

$$R^* = \left( 1 + \frac{EIS_{ii}}{LR_{EA}} \right) \left( 1 + \frac{EIS_{jj}}{LR_{EB}} \right) - \left( \frac{EI}{L} \right)^2 \frac{S_{ij}^2}{R_{EA} R_{EB}}$$